

Finite hyper-elastodynamics

$$(S) \Leftrightarrow (W) \simeq (G) \Leftrightarrow (M)$$

↑ ODE's

$$\sigma_{ij,j} + \rho b_i = 0$$

↑ D'Alembert principle. $b \leftarrow b - a$

$$\int_{\mathcal{V}} w_i \rho a_i + w_{(i,j)} \sigma_{ij} dx = \int_{\mathcal{V}} w_i b_i \rho dx + \int_{\mathcal{A}_h} w_i h_i da$$

pull-back

$$\int_{\mathcal{V}} \rho_0 w_i A_i dx$$

$$A(x,t) = \dot{V}(x,t) = \ddot{U}(x,t)$$

$$\sum_A \sum_B \int_{\mathcal{V}} \rho_0 N_A(x) C_{iA} N_B(x) \ddot{d}_{iB} dx \quad U_i^h(x,t) = \sum_B N_B(x) d_{iB}(t)$$

$$\sum_{A,B} C_{iA} \int_{\mathcal{V}} \rho_0 N_A(x) N_B(x) dx \delta_{ij} \ddot{d}_{jB}(t)$$

$$M_{iAjB}$$

$$\parallel$$

$$M_{PQ}$$

mass matrix

Remark 1: $M_{iAjB} = M_{jBiA} \Leftrightarrow M_{PQ} = M_{QP}$

M is a constant matrix (independent of time)

M is positive-definite. :

$$C^T M C = \sum_A \sum_B C_{iA} \delta_{ij} \int_V \rho_0 N_A N_B dX C_{jB}$$

$$= \int_V W_i^h W_i^h \rho_0 dX \geq 0$$

and $C^T M C = 0$ implies $W_i^h = 0 \Rightarrow C = 0.$

M has the same banded structure as $K.$

The ODE problem:

$$M \ddot{d}(t) + N(d(t)) = F(t)$$

$$d(0) = d_0 \leftarrow$$

$$\dot{d}(0) = v_0 \leftarrow \begin{array}{l} \text{given initial data projected} \\ \text{to the discrete} \\ \text{space} \end{array}$$

The generalized- α method

$$d_n \approx d(t_n) \quad \dot{d}_n \approx v(t_n) \quad a_n \approx a(t_n)$$

determine $d_{n+1}, \dot{d}_{n+1}, a_{n+1}$, s.t.

$$M a_{n+\alpha_m} + N(d_{n+\alpha_f}) = F_{n+\alpha_f} = F(t_{n+\alpha_f})$$

$$d_{n+\alpha_f} = (1 - \alpha_f) d_n + \alpha_f d_{n+1}$$

$$\dot{d}_{n+\alpha_f} = (1 - \alpha_f) \dot{d}_n + \alpha_f \dot{d}_{n+1}$$

$$a_{n+\alpha_m} = (1 - \alpha_m) a_n + \alpha_m a_{n+1}$$

$$d_{n+1} = d_n + \Delta t v_n + \frac{\Delta t^2}{2} [(1-2\beta)a_n + 2\beta a_{n+1}]$$

$$v_{n+1} = v_n + \Delta t [(1-\nu)a_n + \nu a_{n+1}]$$

4 parameters define the algorithm: $\alpha_f, \alpha_m, \beta, \nu$.

Remark 1: We need a_0 to start the algorithm, and

$$M a_0 = F(t_0) - N(d_0).$$

Remark 2: For linear elastodynamics, analysis results have been pursued by Chung & Hulbert (Journal of Applied Mechanics, 1993, Vol 60, pp. 371-375).

• Second-order time accuracy is achieved if

$$\nu = \frac{1}{2} - \alpha_f + \alpha_m, \quad \beta = \frac{1}{4} (1 - \alpha_f + \alpha_m)^2$$

• Unconditional stability:

$$\alpha_m \geq \alpha_f \geq \frac{1}{2}.$$

• High frequency damping

$$\alpha_m = \frac{2 - \rho_\infty}{1 + \rho_\infty}$$

$$\alpha_f = \frac{1}{1 + \rho_\infty}$$

$$\ddot{u} + 2\zeta\omega\dot{u} + \omega^2 u = f \quad \left(\begin{array}{l} (K - \omega^2 M) \psi = 0 \\ M \ddot{u} + C \dot{u} + K u = F \end{array} \right)$$

\swarrow damping ratio $\zeta = (\frac{a}{\omega} + b\omega)/2$ \uparrow undamped frequency \searrow viscous damping $aM + bK$

discretize $\rightarrow X_{n+1} = A X_n \quad X_n = \{d_n, \Delta t v_n, \Delta t^2 a_n\}^T$

$$\rho = \max(|\lambda_1|, |\lambda_2|, |\lambda_3|) \quad \text{spectral radius.}$$

and p is a function of ω st.



Implementation: Given d_n, v_n, a_n

Predictor: $v_{nt+1}^{(0)} = v_n$

$$a_{nt+1}^{(0)} = \frac{\nu-1}{\nu} a_n$$

$$d_{nt+1}^{(0)} = d_n + \Delta t v_n + \frac{\Delta t^2}{2} \left((1-2\beta) a_n + 2\beta a_{nt+1}^{(0)} \right)$$

multi-corrector: repeat the following steps for $i=1, 2, \dots, i_{max}$

$$d_{nt+\alpha_f}^{(i)} = (1-\alpha_f) d_n + \alpha_f d_{nt+1}^{(i)}$$

evaluate at intermediate steps. $v_{nt+\alpha_f}^{(i)} = (1-\alpha_f) v_n + \alpha_f v_{nt+1}^{(i)}$

$$a_{nt+\alpha_m}^{(i)} = (1-\alpha_m) a_n + \alpha_m a_{nt+1}^{(i)}$$

• Form the residual at intermediate solutions

$$R_{nt+1}^{(i)} := -M a_{nt+\alpha_m}^{(i)} - N(d_{nt+\alpha_f}^{(i)}) + F_{nt+\alpha_f}$$

and tangent ~~vector~~ matrix

$$DR_{nt+1}^{(i)} = \frac{\partial R}{\partial a_{nt+\alpha_m}} \frac{\partial a_{nt+\alpha_m}}{\partial a_{nt+1}} \rightarrow \alpha_m$$

$$+ \frac{\partial R}{\partial v_{nt+\alpha_f}} \frac{\partial v_{nt+\alpha_f}}{\partial v_{nt+1}} \frac{\partial v_{nt+1}}{\partial a_{nt+1}} \rightarrow \nu \Delta t$$

$$+ \frac{\partial R}{\partial d_{nt+\alpha_f}} \frac{\partial d_{nt+\alpha_f}}{\partial d_{nt+1}} \frac{\partial d_{nt+1}}{\partial a_{nt+1}} \rightarrow \beta \Delta t^2$$

$\downarrow \alpha_f$

$$DR_{nt+1}^{(i)} = -\alpha_m M + \alpha_f \nu \Delta t \frac{\partial R}{\partial v_{nt+1}} - \alpha_f \beta \Delta t^2 DN(d_{nt+1}^{(i)})$$

- Solve the linear system

$$DR_{nt+1}^{(i)} \cdot \Delta a_{nt+1}^{(i+1)} = R_{nt+1}^{(i)}$$

- update the solution

$$a_{nt+1}^{(i+1)} = a_{nt+1}^{(i)} + \Delta a_{nt+1}^{(i+1)}$$

$$v_{nt+1}^{(i+1)} = v_{nt+1}^{(i)} + \nu \Delta t \Delta a_{nt+1}^{(i+1)}$$

$$d_{nt+1}^{(i+1)} = d_{nt+1}^{(i)} + \beta \Delta t^2 \Delta a_{nt+1}^{(i+1)}$$

- Test if $\|R_{nt+1}^{(i+1)}\| < \text{tol}_A$ or $\|R_{nt+1}^{(i+1)}\| < \text{tol}_R \|R_{nt+1}^{(0)}\|$.

Remark: For linear problems with viscous damping,

$$R_{nt+1}^{(i)} = F_{nt+1} - M a_{nt+1}^{(i)} - C v_{nt+1}^{(i)} - K d_{nt+1}^{(i)}$$

$$DR_{nt+1}^{(i)} = -\alpha_m M - \alpha_f \nu \Delta t C - \alpha_f \beta \Delta t^2 K$$

Remark: $\alpha_m = \alpha_f = 1$: Newmark (should not be used)
 $\alpha_m = 1$: HHT- α

Remark: undamped eigenproblem

$$(K - \lambda M) \psi = 0$$

$$0 \leq \lambda_1 \leq \lambda_2 \leq \dots \leq \lambda_{neg}$$

$$\psi_r^T M \psi_m = \delta_{rm} \quad (10)$$

$$\Rightarrow \{ \lambda_c, \psi_c \}_{c=1}^{n_{eq}} \quad \omega_c = (\lambda_c)^{1/2} \quad \{ \psi_c \} \text{ basis of } \mathbb{R}^{n_{eq}}$$

$$0 = \psi_m^T \left(M \sum_{n=1}^{n_{eq}} \psi_n \ddot{d}_n(t) + C \sum_{n=1}^{n_{eq}} \psi_n \dot{d}_n(t) + K \sum_{n=1}^{n_{eq}} \psi_n d_n(t) - F \right)$$

$$= \ddot{d}_m + 2 \xi \omega \dot{d} + \omega^2 d - F_{\#m} \quad \text{SDOF problem}$$

$$\psi_m^T M \psi_n = \delta_{mn} \quad \psi_m^T K \psi_n = \lambda_n \delta_{mn}$$

$$C = aM + bK \quad \text{Rayleigh damping}$$

$$\psi_m^T C \psi_n = (a + b \lambda_n) \delta_{mn}$$

\uparrow
 foundation of
 numerical analysis
 for linear dynamics
 \uparrow
 Hughes FEM book
 Chap. 9.

Energy analysis of linear problems

Consider a physical problem discretized by the mid-point rule : $\rho_{\infty} = 1$ (i.e. no high frequency damping)

$$M a_{n+\frac{1}{2}} + C v_{n+\frac{1}{2}} + K d_{n+\frac{1}{2}} = F_{n+\frac{1}{2}}$$

$$(\cdot)_{n+\frac{1}{2}} = \frac{1}{2} (\cdot)_n + \frac{1}{2} (\cdot)_{n+1}$$

$$v_{n+1} = v_n + \frac{\Delta t}{2} (a_n + a_{n+1}) = v_n + \Delta t a_{n+\frac{1}{2}}$$

$$d_{n+1} = d_n + \Delta t v_n + \frac{\Delta t^2}{2} a_{n+\frac{1}{2}}$$

$$\begin{aligned}
V_{n+\frac{1}{2}} \cdot M a_{n+\frac{1}{2}} &= V_{n+\frac{1}{2}} \cdot M \frac{V_{n+1} - V_n}{\Delta t} \\
&= \frac{1}{2\Delta t} (V_{n+1} \cdot M V_{n+1} - V_n \cdot M V_n) \\
&= \frac{1}{\Delta t} (K_{n+1} - K_n) \quad K(V_{n+1})
\end{aligned}$$

$V_{n+\frac{1}{2}} \cdot C V_{n+\frac{1}{2}} := \mathcal{D} \geq 0$ dissipation from viscous damping

$$\begin{aligned}
V_{n+\frac{1}{2}} \cdot K d_{n+\frac{1}{2}} &= \frac{d_{n+1} - d_n}{\Delta t} \cdot K d_{n+\frac{1}{2}} \\
&= \frac{1}{2\Delta t} (d_{n+1} \cdot K d_{n+1} - d_n \cdot K d_n) \\
&= \frac{1}{\Delta t} \cancel{\text{something}} (U(d_{n+1}) - U(d_n))
\end{aligned}$$

$$\Rightarrow K(V_{n+1}) + U(d_{n+1}) = K(V_n) + U(d_n) - \Delta t \mathcal{D}$$

$\Rightarrow V_{n+1}$ & d_{n+1} are bounded. \Rightarrow stability.

If we return to nonlinear problems: discretized by mid-point:

$$\int_V \rho_0 w_i A_i dx + \int_V w_{i,I} P_{iI} dx = \int_V w_i \rho_0 B_i dx$$

$$A_i = \dot{V}_i = \ddot{U}_i \quad + \int_{A_H} w_i H_i dA.$$

$\frac{v_{n+1} - v_n}{\Delta t}$ } discretize

• Total linear momentum $L := \int_V \rho_0 V dx$

Pick $w = \xi$ constant vector.

$$\Rightarrow \int_V \rho_0 \xi_i \dot{V}_i dx + \int_V \xi_{i,I} P_{iI} dx = \int_V \xi_i \rho_0 B_i dx + \int_{A_H} \xi_i H_i dA$$

$$\Rightarrow \xi_i \left\{ \frac{D}{Dt} \int_V \rho_0 V_i dx - \int_V \rho_0 B_i dx - \int_{A_H} H_i dA \right\} = 0$$

$$\Rightarrow \boxed{\frac{D}{Dt} L = \int_V \rho_0 B dx + \int_{A_H} H dA}$$

Conservation of linear mom.

Conditional in the sense that constant vectors have to be admissible test function.

no essential BC.

$$\frac{L_{n+1} - L_n}{\Delta t} = \int_V \rho_0 B_{n+1/2} dx + \int_V H_{n+1/2} dA. \quad (104)$$

• Total angular momentum. $J = \int_V \rho_0 \mathcal{L}_i \times \mathbf{v} dX$

$$J_i = \int_V \rho_0 \epsilon_{ijk} \varphi_j v_k dX.$$

Pick $\mathbf{W} = \mathcal{F} \times \varphi_{n+1/2}$.

$$\int_V \rho_0 W_i A_i dX = \int_V \rho_0 \epsilon_{ijk} \mathcal{F}_j x_k \frac{v_{n+1, i} - v_{n, i}}{\Delta t} dX$$

" ϵ_{jki}

$$= \mathcal{F}_j \int_V \rho_0 \epsilon_{jki} x_k \frac{v_{n+1, i} - v_{n, i}}{\Delta t} dX$$

$$= \mathcal{F} \cdot (J_{n+1} - J_n) / \Delta t.$$

Similarly,

$$\int_V \rho_0 \mathcal{F} \times \varphi_{n+1/2} B_{n+1/2} dX = \mathcal{F} \cdot \int_V \varphi_{n+1/2} \times B_{n+1/2} dX.$$

$$\int_{A_H} \mathcal{F} \times \varphi_{n+1/2} H_{n+1/2} dA = \mathcal{F} \cdot \int_{A_H} \varphi_{n+1/2} \times H_{n+1/2} dA.$$

Skew tensor and its axial vector.

$$\mathbf{W}_{\mathcal{F}} \mathbf{u} = \mathcal{F} \times \mathbf{u}$$

Skew tensor: $W_{ij} = -\epsilon_{ijk} \mathcal{F}_k$

axial/dual vector: $\mathcal{F}_k = -\frac{1}{2} \epsilon_{ijk} W_{ij}$

$$\begin{aligned}
\int_V (\xi^k \varphi_{n+1/2, i})_{,I} P_{iI} dx &= \int_V W_{\xi} \epsilon_{ij} \varphi_{n+1/2, j, I} P_{iI} dx \\
&= \int_V W_{\xi} \epsilon_{ij} F_{jI} P_{iI} dx \\
&= \underbrace{W_{\xi} \epsilon_{ij}}_{\text{Skew}} \underbrace{\int_V F_{iJ} S_{JI} F_{jI} dx}_{\text{Symm.}} \\
&= 0
\end{aligned}$$

$$\Rightarrow \xi \cdot \left\{ \frac{J_{n+1} - J_n}{\Delta t} - \int_V \varphi_{n+1/2} \times B_{n+1/2} dx - \int_{A_H} \varphi_{n+1/2} \times H_{n+1/2} dA \right\} = 0$$

$$\Rightarrow \boxed{\frac{J_{n+1} - J_n}{\Delta t} = \int_V \varphi_{n+1/2} \times B_{n+1/2} dx + \int_{A_H} \varphi_{n+1/2} \times H_{n+1/2} dA}$$

resultant moment.

- 1) $\xi \times x$ needs to be admissible.
- 2) Stress needs to be symmetric.

• Total energy (Hamiltonian)

Pick $w_i = v_{i, n+1/2}$ ← G data is time independent.

$$\hookrightarrow \mathfrak{S}_U = \{ U_k : \dots U_k(\cdot, t) = G \text{ on } A_G \}$$

$$\mathfrak{S}_V = \{ v_n : \dots v_n(\cdot, t) = \dot{G} \text{ on } A_G \}$$

$$\int_V \rho_0 v_{i, n+1/2} (v_{n+1} - v_n) / \Delta t + v_{i, n+1/2, I} P_{iI} dX$$

$$= \int_V v_i \rho_0 B_i dX + \int_{A_H} v_i H_i dA$$

$$\textcircled{1} = \int_V \frac{\rho_0}{2\Delta t} (|v_{n+1}|^2 - |v_n|^2) dX$$

$$\textcircled{2} = \int_V v_{i, n+1/2, I} F_{iJ, n+1/2} S_{JI} dX$$

$$v_{i, n+1/2} = \frac{U_{i, n+1} - U_{i, n}}{\Delta t}$$

$$v_{i, n+1/2, I} = \frac{1}{\Delta t} (F_{iI, n+1} - F_{iI, n})$$

$$F_{iJ, n+1/2} v_{i, n+1/2, I} = \frac{1}{2\Delta t} (C_{n+1} - C_n)_{IJ}$$

$$\textcircled{2} = \int_V \frac{1}{2\Delta t} (C_{n+1} - C_n) : S dX$$

In general, $\frac{1}{2}(C_{n+1} - C_n) : S_{n+1/2} \neq \Phi(C_{n+1}) - \Phi(C_n)$

Simo: Determine a collocation parameter θ such that

$$\frac{1}{2}(C_{n+1} - C_n) : S(C_{n+\theta}) = \Phi(C_{n+1}) - \Phi(C_n)$$

and θ is d. solved from the above eqn. at each quadrature pt.

See, Simo & Tarnow ZAMP 1992, 43: 757-792
 Laurson & Meng, CMAME 2001, 190: 6309-6322.

Gonzalez. discrete gradient CMAME, 2000, 190: 1763-1783.

$$S_{alg} := S(C_{n+1/2}) + \frac{\Phi(C_{n+1}) - \Phi(C_n) - S(C_{n+1/2}) : Z_n}{\|Z_n\|^2} Z_n$$

$$Z_n := (C_{n+1} - C_n) / 2$$

$$Z_n : S_{alg} = \Phi(C_{n+1}) - \Phi(C_n)$$

directionality property

S_{alg} is symm.

angular mom. conservation.

- enhancement.
- $O(\|Z_n\|^2)$
do not destroy the temporal accuracy.

Replace S by S_{alg} in the ~~momentum~~ weak-form problem,

one has

$$\begin{aligned} & \frac{1}{\Delta t} \int_V \frac{\rho_0}{2} |v_{n+1}|^2 + \Phi(c_{n+1}) dx \\ & - \frac{1}{\Delta t} \int_V \frac{\rho_0}{2} |v_n|^2 + \Phi(c_n) dx \\ & = \int_V v_{n+1/2} \cdot \rho_0 B dx + \int_{A_H} v_{n+1/2} \cdot H dA \end{aligned}$$

Energy preservation / stability.

Remark: High-mode dissipation can be added. See, e.g.
Armero & Romero, *CMAME*, 2001, 190: 2603-2649.