

Stress tensors :

$$t_i = \sigma_{ij} n_j \quad \text{or} \quad \vec{t} = \bar{\sigma} \vec{n}$$

→ Cauchy stress (true stress): a linear operator that transforms unit normal vector of an area element to the traction acting on the surface.

- $\sigma_{ij} = \sigma_{ji}$ due to the balance of angular momentum (without couple-stresses).
- $\bar{\tau} = J \bar{\sigma}$ Kirckhoff stress tensor

Now, we introduce the traction on the corresponding initial configuration T_i satisfying $T_i dA = t_i da$

→ T_i and t_i are parallel. different due to the scaling of areas.

Introduce a two-point tensor P_{iI} such that

$$T_i = P_{iI} N_I$$

→ The first Piola-Kirckhoff stress (PK)

$$P_{iI} \underline{N_I} dA = \sigma_{ij} n_j da = \sigma_{ij} J \bar{F}_{Ij}^{-1} \underline{N_I} dA$$

$$P_{iI} = J \sigma_{ij} \bar{F}_{Ij}^{-1} = J \sigma_{ij} (\bar{F}^{-T})_{jI}$$

or

$$\bar{P} = J \bar{\sigma} \bar{F}^{-T} = \bar{\tau} \bar{F}^{-T}$$

Let $V = B_\epsilon$: a spherical ball with radius $\epsilon > 0$

$$\psi = \varphi_t(B_\epsilon) = b_\epsilon$$

$$S = \partial b_\epsilon$$

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$$\int_{S=\partial b_\epsilon} \vec{t} da = \int_{S=\partial B_\epsilon} \vec{T} dA$$

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$$\int_S \sigma n da$$

$$\int_S P N dA$$

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$$\int_{\partial b} \sigma_{ij,j} dv$$

$$\int_V P_{iI,I} dV$$

||

$$\int_V \sigma_{ij,j} J dV$$

⇒ localization by shrinking the ball :

$$\sigma_{ij,j} J = P_{iI,I}$$

or

$$J \operatorname{div} \sigma = \operatorname{DIV} P$$

S is a symmetric material tensor defined as

$$S = \bar{F}^{-1} P = J \bar{F}^{-1} \sigma \bar{F}^{-T} = \bar{F}^{-1} \tau \bar{F}^{-T}$$

or

$$S_{IJ} = \bar{F}_{Ii}^{-1} P_{iJ} = J \bar{F}_{Ii}^{-1} \sigma_{ij} \bar{F}_{Jj}^{-1} = \bar{F}_{Ii}^{-1} \tau_{ij} \bar{F}_{Jj}^{-1}$$

Constitutive Relations:

There exists a strain energy density function per unit undeformed volume:

$$\begin{aligned} \Phi &= \Phi(E) \quad \leftarrow \frac{1}{2}(C-I) = \frac{1}{2}(F^T F - I) \\ &= \hat{\Phi}(C) = \tilde{\Phi}(F) \end{aligned}$$

Constitution:

$$S = \frac{\partial \Phi}{\partial E} \quad \text{or} \quad S_{IJ} = \frac{\partial \Phi}{\partial E_{IJ}}$$

$$C = \frac{\partial S}{\partial E} = \frac{\partial^2 \Phi}{\partial E \partial E} \quad C_{IJKL} = \frac{\partial S_{IJ}}{\partial E_{KL}} = \frac{\partial^2 \Phi(E)}{\partial E_{IJ} \partial E_{KL}}$$

rank-four tensor,
elasticity tensor

→ spatial counterpart:

$$c_{ijke} = \bar{J}^{-1} F_{iI} F_{jJ} F_{kK} F_{eL} C_{IJKL}$$

↑
push-forward of rank-4 tensor

Ex. St. Venant - Kirchhoff material.

$$\Phi(E) = \frac{1}{2} \lambda (\text{tr} E)^2 + \mu E : E$$

$$\frac{\partial \text{tr} E}{\partial E} = I, \quad \frac{\partial E : E}{\partial E} = 2E$$

$$S = \frac{\partial \Phi}{\partial E} = \lambda \text{tr} E I + 2\mu E$$

$$C = \frac{\partial S}{\partial E} = \lambda I \otimes I + 2\mu \mathbb{I}$$

or

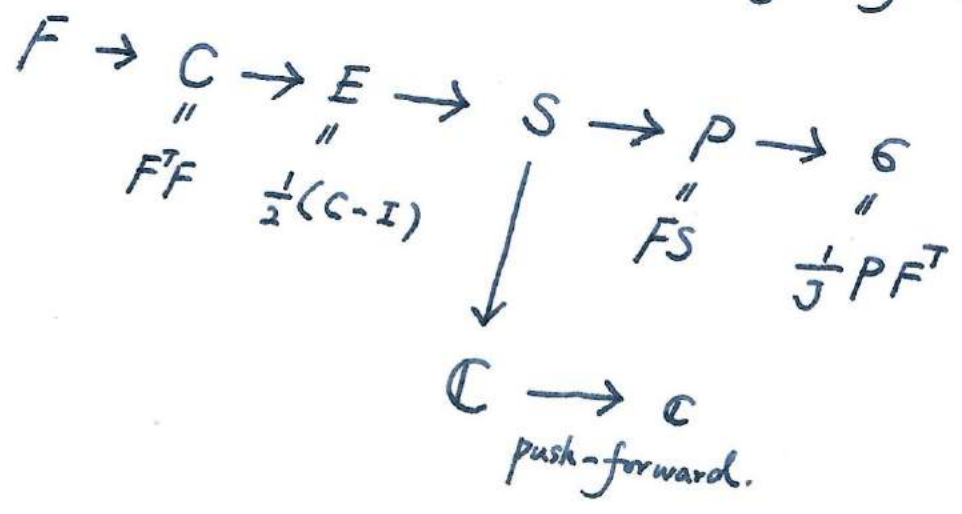
$$C_{IJKL} = \lambda \delta_{IJ} \delta_{KL} + 2\mu (\delta_{IK} \delta_{JL} + \delta_{IL} \delta_{JK})$$

Remark: 1: oftentimes, we write strain energy function in terms of C. and

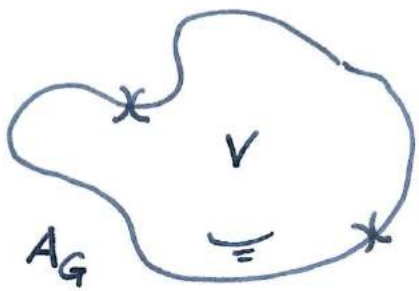
$$S = 2 \frac{\partial \Phi(C)}{\partial C}$$

$$C = 4 \frac{\partial^2 \Phi}{\partial C \partial C}$$

2: in the material model, the input is typically F.



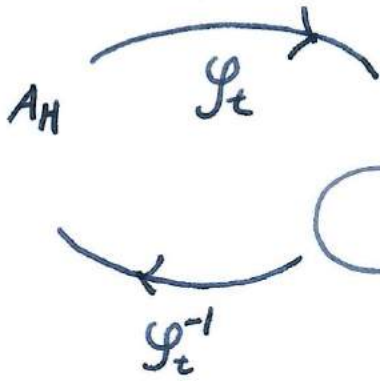
Boundary-value problem



~~$U = G$~~ on A_G

$T = PN = H$ on A_H

~~Φ_t~~



$\psi = \Phi_t(V)$

$u = g$ on $a_g = \Phi_t(A_G)$

$t = \sigma n = \frac{h}{\#}$ on $a_h = \Phi_t(A_H)$

$\Phi_t(A_H)$
||
 a_h

on the current configuration:

$\text{div } \sigma + f = 0$ in ψ

or $\sigma_{ij,j} + f_i = 0$

body force per unit undeformed volume

$f_i = \rho b_i$ acceleration.

mass per deformed volume

$\int_{b_\epsilon = \Phi_t(B_\epsilon)} \rho(x, t) \, d\psi = \text{const}$
|| $\int J \, dV$

$\int_{B_\epsilon} \rho_0(x) \, dV$

$\rho(x, t) J(x, t) = \rho_0(x)$ or $\rho_t \circ \Phi_t J = \rho_0$

or $\rho J = \rho_0$ Local Lagrangian form of mass conservation.

$$J (\operatorname{div} \sigma + \rho b) = 0$$

$$\parallel$$

$$\operatorname{DIV} P + \rho_0 B \longleftarrow B(x, t) = b(\varphi_i(x), t)$$

Weak-form problem:

$$\int_V w_{i,j} \sigma_{ij} dV = \int_V w_i \rho b_i dV + \int_{A_h} w_i h_i da \quad \forall w_i \in \mathcal{V}_i$$

$$W(x) = w(\varphi_i(x)) \rightarrow w_{i,j} F_{jI} = W_{i,I} \quad \begin{matrix} h_i da \\ \parallel \\ H_i dA \end{matrix}$$

$$\int_V W_{i,I} \bar{F}_{Ij}^{-1} \sigma_{ij} J dV = \int_V w_i \rho b_i J dV + \int_{A_H} w_i H_i dA$$

$$\int_V W_{i,I} P_{iI} dV \quad \int_V w_i \rho_0 B_i dV$$

Remark: $h_i da = H_i dA$ meaning h_i is parallel to H_i

and scaled by a factor $\frac{da}{dA} = \sqrt{J^2 \bar{F}_{Ii}^{-1} N_i \bar{F}_{Ij}^{-1} N_j}$

Remark: if B, G, H do not vary with $= J \sqrt{N \cdot \bar{C}^{-1} N}$.

the deformation, we call them "dead loads."

If we define $\mathcal{F}(\varphi) := \int_V w_{i,I} P_{iI} dv - \int_V w_i \rho_0 B_i dv$

then the Newton-Raphson method will be $-\int_{A_H} w_i H_i dA$

$$D\mathcal{F}(\varphi) \cdot \Delta U = -\mathcal{F}(\varphi)$$

↑
Linear action

Can be calculated by the Gateaux derivative:

$$\lim_{\varepsilon \rightarrow 0} \left\{ \frac{d}{d\varepsilon} \mathcal{F}(\varphi + \varepsilon \Delta U) \right\}$$

$$\mathcal{F}(\varphi + \varepsilon \Delta U) = \int_V w_{i,I} \frac{\partial}{\partial X_J} (\varphi_i + \varepsilon \Delta U_i) S_{JI} (E(\varphi + \varepsilon \Delta U)) dv$$

$$\frac{1}{2} \left\{ \frac{\partial}{\partial X} (\varphi + \varepsilon \Delta U)^T \frac{\partial}{\partial X} (\varphi + \varepsilon \Delta U) - I \right\}$$

$$\lim_{\varepsilon \rightarrow 0} \mathcal{F}(\varphi + \varepsilon \Delta U) = \int_V w_{i,I} \overset{\textcircled{1}}{\varphi_i} S_{JI} + w_{i,I} F_{iJ} \frac{d}{d\varepsilon} \Big|_{\varepsilon=0} S_{JI} (E(\varphi + \varepsilon \Delta U)) \overset{\textcircled{2}}{dx}$$

$$\frac{d}{d\varepsilon} S_{JI} (E(\varphi + \varepsilon \Delta U)) = \frac{\partial S_{JI}}{\partial E_{KL}} \left\{ \frac{1}{2} \Delta U_{k,K} F_{KL} + \frac{1}{2} \Delta U_{k,L} F_{kK} \right\}$$

$$\begin{aligned} \textcircled{1} : \int_V w_{i,I} \Delta U_{i,J} S_{JI} dv &= \int_V w_{i,j} F_{jI} \Delta u_{i,e} F_{eJ} S_{JI} \bar{J}^{-1} d\bar{v} \\ &= \int_V w_{ij} \sigma_{je} \Delta u_{i,e} d\bar{v} = \int_V w_{ij} \underbrace{\sigma_{je} \delta_{ik}}_{d_{ijk}} u_{k,e} d\bar{v} \end{aligned}$$

$d_{ijkl} = d_{klij} = \sigma_{ij} \delta_{ki} \leftarrow$ has major symmetry

\hookrightarrow minor symmetry is lost: $d_{1212} \neq d_{2121}$

\rightarrow it can see the skew part in w_{ij} and behaves as resistant to rotation

\rightarrow Stiffness due to the current stress state, known as the geometrical stiffness.

$$\textcircled{2}: \int_V w_{i,I} F_{iJ} C_{JIKL} \frac{1}{2} (\Delta U_{k,K} F_{kL} + \Delta U_{k,L} F_{kK}) dV$$

$$= \int_V \underbrace{w_{i,j}}_{\varphi_i(V)} F_{jI} F_{iJ} C_{JIKL} \underbrace{\Delta U_{k,K} F_{kL}}_{\Delta u_{k,l} F_{kL}} J^{-1} dV$$

$$= \int_V w_{i,j} \underbrace{(J^{-1} F_{iI} F_{jJ} F_{kK} F_{lL} C_{JIKL})}_{C_{ijkl}} \Delta u_{k,l} dV$$

major symm: $C_{klij} = J^{-1} F_{kI} F_{lJ} F_{iK} F_{jL} C_{IJKL}$
 $= J^{-1} F_{kI} F_{lJ} F_{iK} F_{jL} C_{KLIJ}$
 $= C_{ijkl}$

minor symm: $C_{jike} = \bar{J}^{-1} F_{jI} F_{iJ} F_{kK} F_{eL} C_{IKL} = C_{ijke}$

$$C_{ijke} = C_{ijek}$$

$$\int_U w_{i,j} C_{ijke} \Delta u_{k,e} dv = \int_U w_{(i,j)} C_{ijke} \Delta u_{(k,e)} dv.$$

① + ②: $\int_U w_{i,j} a_{ijke} \Delta u_{k,e} dv$

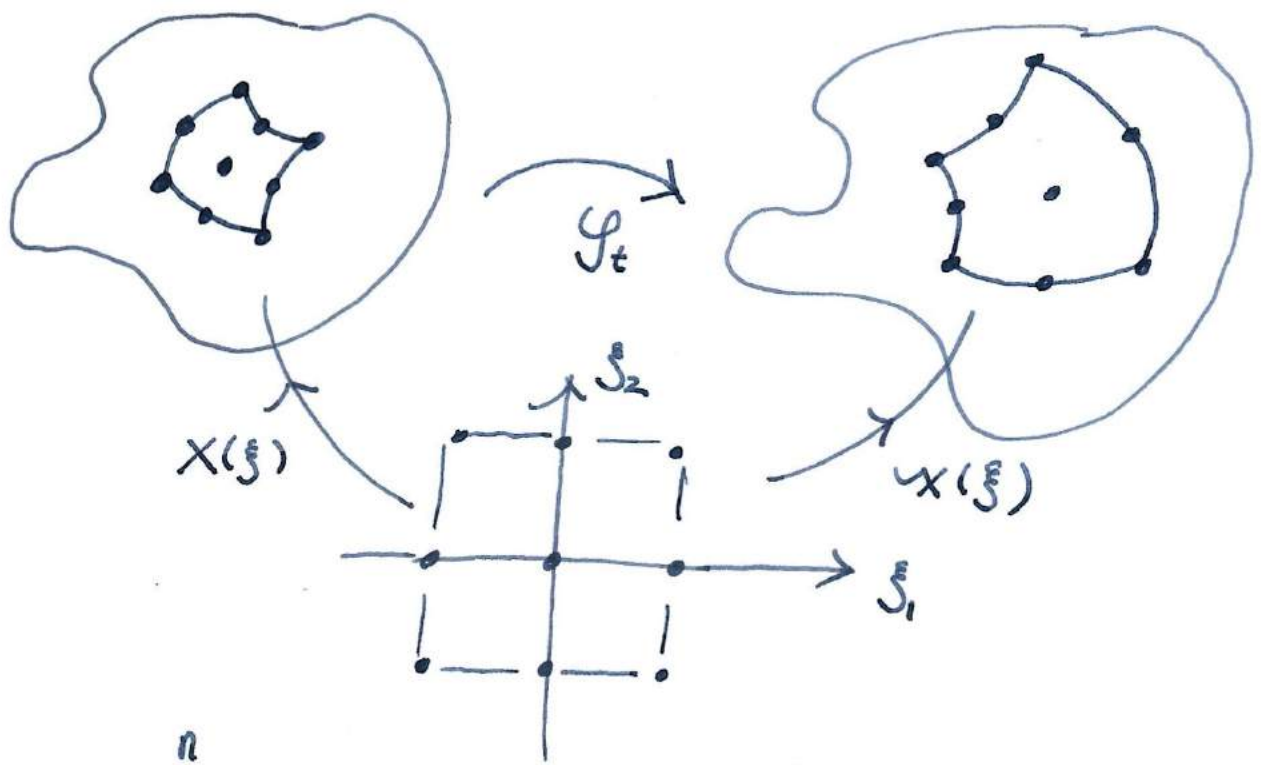
$\rightarrow \bar{a} = \bar{c} + \bar{d}$ has major but no minor symmetry

pull-back

$$\delta_{ij} S_{IJ} + F_{ik} F_{jL} C_{IKL} = J F_{Ik}^{-1} F_{jL}^{-1} a_{ikjl}$$

$$= \int_V w_{i,I} A_{iIjJ} \Delta u_{j,J} dv$$

One can show $A_{iIjJ} = \frac{\partial^2 \phi}{\partial F_{iI} \partial F_{jJ}}$.



$$X_I = \sum_{a=1}^n N_a(\xi) X_{ia}^e$$

$$X_i = \dots \dots X_{ia}^e$$

$$\varphi_i = \dots \dots (\cancel{X_{ia}^e})$$

$$U_i = \dots \dots U_{ia}^e$$

$$u_i = \dots \dots u_{ia}^e$$

$$\int_{V^e} \dots J dV = \int_{v^e} \dots dve$$

$$\int_{\square} \dots J \det \left(\frac{\partial X}{\partial \xi} \right) d\xi_1 d\xi_2 = \int_{\square} \dots \det \left(\frac{\partial X}{\partial \xi} \right) d\xi_1 d\xi_2$$

det F

$$F_{iI}^e = \frac{\partial \varphi_i}{\partial X_I} = \frac{\partial X_i^e}{\partial X_I} = \frac{\partial X_i^e}{\partial \xi_\alpha} \frac{\partial \xi_\alpha}{\partial X_I}$$

$$= \frac{\partial X_i^e}{\partial \xi_\alpha} \left[\left(\frac{\partial X}{\partial \xi} \right)^{-1} \right]_{\alpha I}$$

$$DF(\varphi) \cdot \Delta u = -\mathcal{F}(\varphi) = \int_{v=\varphi(v)} w_i \rho b_i \, dv + \int_{a_h} w_i h_i \, da$$

$$\int_{v=\varphi(v)} w_{(i,j)} c_{ijke} \Delta u_{(k,e)} \, dv - \int_v w_{(i,j)} \sigma_{ij} \, dv$$

$$\int_v w_{ij} \delta_{ik} \sigma_{je} \Delta u_{k,e} \, dv$$

$$\begin{aligned} \textcircled{1} : K_{pq}^e &= K_{iajb}^e = e_i^T \int_v B_a^T D B_b \, dv e_j \\ &= e_i^T \int_{\square} B_a^T D B_b \det\left(\frac{\partial X}{\partial \xi}\right) \underbrace{d\xi_1 d\xi_2 d\xi_3}_{d\xi} e_j \end{aligned}$$

$$\textcircled{2} : K_{pq}^e = K_{iajb}^e = \delta_{ij} K_{ab}^e \quad K_{ab}^e = \int_v N_{a,i} \sigma_{ij} N_{b,j} \, dv$$

$$\textcircled{3} : \int_{\square} \rho N_a^{(\xi)} b_i \det\left(\frac{\partial X}{\partial \xi}\right) d\xi$$

$\uparrow \quad \quad \quad \uparrow$
 $J^{-1} \rho_0 \quad b_i(X(\xi))$

$$\textcircled{4} : \int_{a_h} N_a h_i \, da = \int_{\partial \square} N_a h_i(X(\xi)) \frac{|da|}{|d\xi|} d\xi$$

$$\textcircled{5} : e_i^T \int_{\square} B_a^T \sigma^{\text{vect}} \det\left(\frac{\partial X}{\partial \xi}\right) d\xi$$

\uparrow
 Voigt notation.